

## Position/Force Control of a Walking Robot\*

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**Abstract:** This paper analyzes the performance of a classical position *PD* algorithm and a new cascade controller, involving position and force feedback loops, for multi-legged locomotion systems. For that objective the robot prescribed motion is characterized in terms of several locomotion variables. A group of indices measures the walking performance based on the robot and terrain dynamical properties and on the robot hip trajectories. A set of experiments reveals the performance of the control architectures in the viewpoint of the proposed indices.

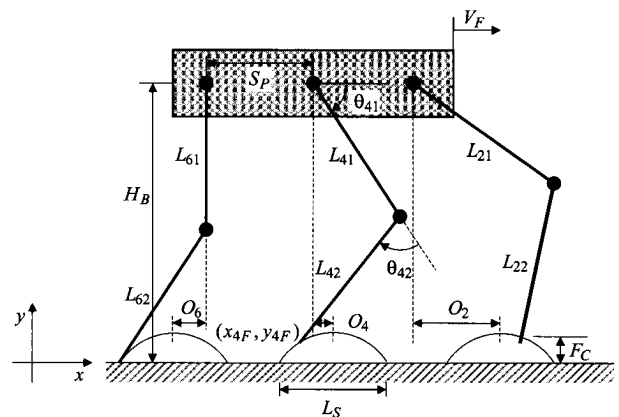
**Keywords:** Walking Robots, Control Algorithms, *PID* Control Algorithm, Cascade Control

### 1. Introduction

**W**ALKING machines allow locomotion in terrain inaccessible to other type of vehicles, since they do not need a continuous support surface [1]. On the other hand, the requirements for leg coordination and control impose difficulties beyond those encountered in wheeled robots [2]. There exists a class of walking machines for which walking is a natural dynamic mode. Once started on a shallow slope, a machine of this class will settle into a steady gait, without active control or energy input [3], [4]. However, the capabilities of these machines are quite limited. Previous studies focused mainly in the control at the leg level and leg coordination using neural networks [5], [6], fuzzy logic [7], central pattern generators [8], [9] and subsumption architecture [10], [11]. There is also a growing interest in using insect locomotion schemes to control walking robots [12]–[15]. In spite of the diversity of approaches, for multi-legged robots the control at the joint level is usually implemented through a simple *PID* scheme with position/velocity feedback [9], [14]. Other approaches include sliding mode control [16], computed torque control [17], hybrid force/position control [18] and fractional order control [19].

With these facts in mind, a simulation model for multi-leg locomotion systems was developed, for several periodic gaits. Based on this tool, the present study compares two different robot control architectures, namely a Proportional-Derivative position algorithm (*PD-P*) and a cascade of a Proportional-Derivative position control with foot force feedback (*PD-P&F*). For the case of the *PD-P&F* it is also implemented an adjusting scheme that varies the gain in the forward force loop according with the step evolution.

The aim of the study is to verify the performance of the two control architectures and the influence of foot force feedback on the system stability and robustness for variable



**Fig. 1** Coordinate system and variables that characterize the motion trajectories of the multi-legged robot

ground characteristics. In this line of thought, the analysis is based on the formulation of a set of indices measuring the robot and ground dynamics as well as the hip trajectory errors during walking.

Several simulations reveal the superior performance of the control architecture with foot force feedback and a gain-adjusting scheme of the forward force loop during the support phase. This control algorithm minimizes the proposed indices, particularly in real situations where we have non-ideal actuators with saturation.

Bearing these facts in mind, the paper is organized as follows. Section 2 introduces the hexapod model and the motion-planning scheme. Sections 3 and 4 present the robot control architecture and formulate the optimizing indices, respectively. After this, Section 5 develops a set of experiments that reveal the performance of the different control architectures and, finally, Section 6 outlines the main conclusions. This paper concludes with an appendix, in Section 7, describing the ground model and the parameter calculation.

### 2. Robot Kinematics and Trajectory Planning

We consider a walking system with  $n$  legs, equally distributed along both sides of the robot body, having each one two rotational joints (**Fig. 1**).

Motion is described by means of a world coordinate sys-

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tem. The kinematic model comprises: the cycle time  $T$ , the duty factor  $\beta$ , the transference time  $t_T = (1 - \beta)T$ , the support time  $t_S = \beta T$ , the step length  $L_S$ , the stroke pitch  $S_P$ , the body height  $H_B$ , the maximum foot clearance  $F_C$ , the  $i$ th leg lengths  $L_{i1}$  and  $L_{i2}$  and the foot trajectory offset  $O_i$  ( $i = 1, \dots, n$ ). Moreover, we consider a periodic trajectory for each foot, with body velocity  $V_F = L_S/T$ .

Given a particular gait and duty factor  $\beta$ , it is possible to calculate for leg  $i$  the corresponding phase  $\phi_i$  and the time instant where each leg leaves and returns to contact with the ground [2].

From these results, and knowing  $T$ ,  $\beta$  and  $t_S$ , the Cartesian trajectories of the tip of the feet must be completed during  $t_T$ . Based on this data, the trajectory generator is responsible for producing a motion that synchronizes and coordinates the legs.

For each cycle the desired trajectory of the foot of the swing leg is computed either through cycloidal (Eq. (1)) or sinusoidal (Eq. (2)) functions. For example, considering that the transfer phase starts at  $t = 0$  s, for leg  $i = 1$  we have the foot trajectory  $\mathbf{p}_{Fd}(t) = [x_{iFd}(t) \ y_{iFd}(t)]^T$ :

- during the transfer phase:

$$\mathbf{p}_{Fd}(t) = \begin{bmatrix} V_F \left[ t - \frac{1}{2\pi f} \sin\left(\frac{2\pi t}{T}\right) \right] \\ \frac{F_C}{2} \left[ 1 - \cos\left(\frac{2\pi t}{T}\right) \right] \end{bmatrix} \quad (1)$$

$$\mathbf{p}_{Fd}(t) = \begin{bmatrix} \left(\frac{t_T}{T}\right) \left[ \left(\frac{L_S t}{t_T}\right) - \frac{L_S}{2\pi} \sin\left(\frac{2\pi t}{T}\right) \right] \\ \frac{2F_C t'}{T} - \frac{F_C}{2\pi} \sin\left(\frac{4\pi t'}{T}\right) \end{bmatrix},$$

$$t' = \begin{cases} t, & 0 \leq t < T/2 \\ T - t, & T/2 \leq t < T \end{cases} \quad (2)$$

- during the stance phase:

$$\mathbf{p}_{Fd}(t) = [V_F T \ 0]^T. \quad (3)$$

The robot body, and by consequence the legs hips, is assumed to have a desired horizontal movement with a constant forward speed  $V_F$ . Therefore, for leg  $i$  the Cartesian coordinates of the hip of the legs are given by  $\mathbf{p}_{Hd}(t) = [x_{iHd}(t) \ y_{iHd}(t)]^T$ :

$$\mathbf{p}_{Hd}(t) = [V_F t \ H_B]^T. \quad (4)$$

Once defined the coordinates of the feet and hips of the robot it is possible to obtain the leg joint positions and velocities using the inverse kinematics  $\psi^{-1}$  and the Jacobian  $\mathbf{J} = \partial\psi/\partial\mathbf{q}$ .

The algorithm for the forward motion planning accepts the desired Cartesian trajectories of the legs feet  $\mathbf{p}_{Fd}(t)$  and hips  $\mathbf{p}_{Hd}(t)$  as inputs and, by means of an inverse kinematics algorithm, generates the related joint trajectories  $\mathbf{q}_d(t) = [\theta_{i1d}(t) \ \theta_{i2d}(t)]^T$ , selecting the solution corresponding to a forward knee:

$$\mathbf{p}_d(t) = [x_{id}(t) \ y_{id}(t)]^T = \mathbf{p}_{Hd}(t) - \mathbf{p}_{Fd}(t) \quad (5)$$

$$\mathbf{p}_d(t) = \psi[\mathbf{q}_d(t)] \Rightarrow \mathbf{q}_d(t) = \psi^{-1}[\mathbf{p}_d(t)] \quad (6)$$

$$\dot{\mathbf{q}}_d(t) = \mathbf{J}^{-1}[\dot{\mathbf{p}}_d(t)]. \quad (7)$$

In order to avoid the impact and friction effects, at the planning phase we estimate null velocities of the feet in the instants of landing and taking off, assuring also the velocity continuity.

### 3. Robot Dynamics and Control Architecture

The planned joint trajectories constitute the reference for the robot control system. The model for the robot inverse dynamics is formulated as:

$$\mathbf{\Gamma} = \mathbf{H}(\mathbf{q})\ddot{\mathbf{q}} + \mathbf{c}(\mathbf{q}, \dot{\mathbf{q}}) + \mathbf{g}(\mathbf{q}) - \mathbf{F}_{RH} - \mathbf{J}_F^T(\mathbf{q})\mathbf{F}_{RF} \quad (8)$$

where  $\mathbf{\Gamma} = [f_{ix} \ f_{iy} \ \tau_{i1} \ \tau_{i2}]^T$  ( $i = 1, \dots, n$ ) is the vector of forces/torques,  $\mathbf{q} = [x_{iH} \ y_{iH} \ \theta_{i1} \ \theta_{i2}]^T$  is the vector of position coordinates,  $\mathbf{H}(\mathbf{q})$  is the inertia matrix and  $\mathbf{c}(\mathbf{q}, \dot{\mathbf{q}})$  and  $\mathbf{g}(\mathbf{q})$  are the vectors of centrifugal/Coriolis and gravitational forces/torques, respectively. The  $n \times m$  matrix  $\mathbf{J}_F^T(\mathbf{q})$  is the transpose of the robot Jacobian matrix,  $\mathbf{F}_{RH}$  is the  $m \times 1$  vector of the body inter-segment forces and  $\mathbf{F}_{RF}$  is the  $m \times 1$  vector of the reaction forces that the ground exerts on the robot feet. Concerning  $\mathbf{F}_{RF}$ , it is considered that these forces are null during the foot transfer phase.

During the system simulation, Eq. (8) is integrated through the Runge-Kutta method [20].

Furthermore, we consider that the joint actuators are not ideal, exhibiting a saturation given by:

$$\tau_{ijm} = \begin{cases} \tau_{ijC}, & |\tau_{ijm}| \leq \tau_{ijMax} \\ \text{sgn}(\tau_{ijC})\tau_{ijMax}, & |\tau_{ijm}| > \tau_{ijMax} \end{cases} \quad (9)$$

where, for leg  $i$  and joint  $j$ ,  $\tau_{ijC}$  is the controller demanded torque,  $\tau_{ijMax}$  is the maximum torque that the actuator can supply and  $\tau_{ijm}$  is the motor effective torque.

Figure 2 presents the dynamic model for the hexapod body and foot-ground interaction.

The contact of the  $i$ th robot feet with the ground is modeled through a non-linear system [21] with damping  $B'_{\eta F}$  and stiffness  $K_{\eta F}$  ( $\eta = \{x, y\}$ ) in the {horizontal, vertical} directions, respectively, according with the equations:

$$\begin{aligned} f_{ixF} &= -K_{xF}\Delta_{ixF} - B'_{xF}(-\Delta_{iyF})\dot{\Delta}_{ixF}, \\ \Delta_{ixF} &= x_{iF} - x_{iF0}, \quad \dot{\Delta}_{ixF} = \dot{x}_{iF} - \dot{x}_{iF0} \end{aligned} \quad (10)$$

$$\begin{aligned} f_{iyF} &= -K_{yF}\Delta_{iyF} - B'_{yF}(-\Delta_{iyF})^v \dot{\Delta}_{iyF}, \\ \Delta_{iyF} &= y_{iF} - y_{iF0}, \quad \dot{\Delta}_{iyF} = \dot{y}_{iF} - \dot{y}_{iF0} \end{aligned} \quad (11)$$

where  $x_{iF0}$  and  $y_{iF0}$  are the coordinates of foot  $i$  touchdown and  $v \approx 1.0$  is a parameter dependent on the ground characteristics [22].

Furthermore, the robot body is divided in  $n$  identical segments (each with mass  $M_b n^{-1}$ ) and a linear spring-dashpot

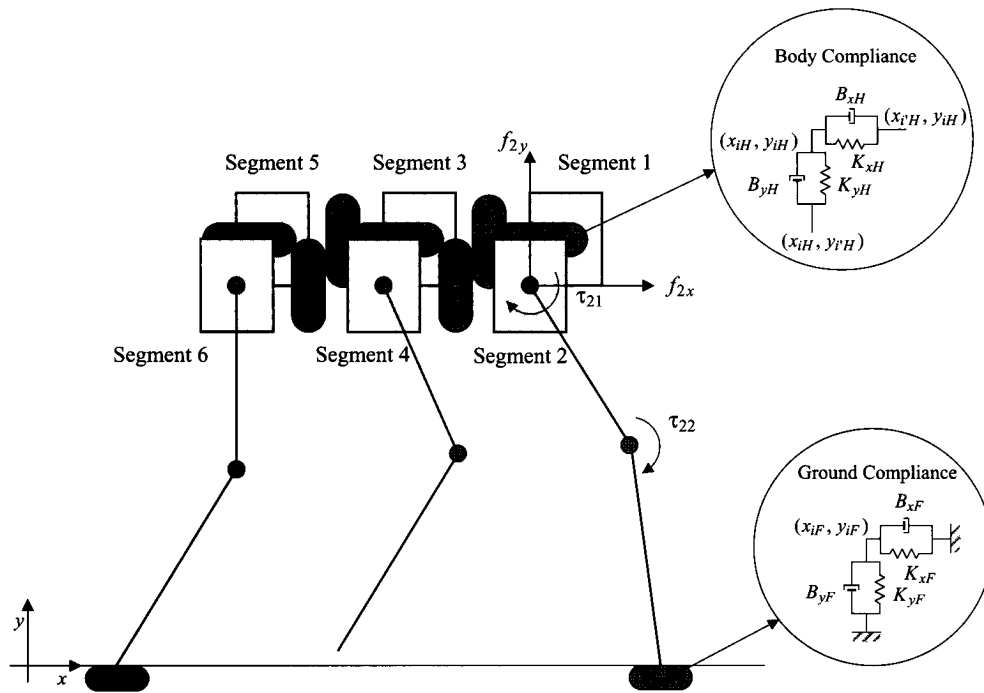


Fig. 2 Model of the robot body and foot-ground interaction

system is adopted to implement the intra-body compliance:

$$f_{ixH} = \sum_{i'=1}^u \left( -K_{xH} \Delta_{i'xH} - B_{xH} \dot{\Delta}_{i'xH} \right), \quad (12)$$

$$\Delta_{i'xH} = x_{iH} - x_{i'H}, \quad \dot{\Delta}_{i'xH} = \dot{x}_{iH} - \dot{x}_{i'H}$$

$$f_{iyH} = \sum_{i'=1}^u \left( -K_{yH} \Delta_{i'yH} - B_{yH} \dot{\Delta}_{i'yH} \right), \quad (13)$$

$$\Delta_{i'yH} = y_{iH} - y_{i'H}, \quad \dot{\Delta}_{i'yH} = \dot{y}_{iH} - \dot{y}_{i'H}$$

where  $(x_{i'H}, y_{i'H})$  are the hip coordinates and  $u$  is the total number of segments adjacent to leg  $i$ , respectively.

The general control architecture of the hexapod robot is presented in Fig. 3. The trajectory planning is held at the Cartesian space but the control is performed in the joint space, which requires the integration of the inverse kinematic model in the forward path. The base algorithm (*PD-P*) considers only a position/velocity feedback and, consequently, the series of  $G_{c1}$  and  $G_{c2}$  can be substituted by a single block. The base architecture is improved with the introduction of a second internal feedback loop with information of the foot-ground interaction force (*PD-P&F*). In this case,  $G_{c1}$  and  $G_{c2}$  are in a cascade structure in the forward control path.

For the controller  $G_{c1}(s)$  we adopt a position/velocity *PD* algorithm:

$$G_{C1j}(s) = Kp_j + Kd_j s, \quad j = 1, 2 \quad (14)$$

where  $Kp_j$  and  $Kd_j$  are the proportional and derivative gains for joint  $j$ , respectively.

For  $G_{c2}$  we consider a simple *P* controller with gain  $Kp_j$  that is varied according to a scheme described in the sequel.

#### 4. Measures for Performance Evaluation

In mathematical terms we establish four global measures of the overall performance of the mechanism in an average sense [23]–[25]. In this perspective, we define three indices  $\{E_{av}, F_L, \tau_P\}$  inspired on the robot dynamics and one index  $\{\varepsilon_{xyH}\}$  based on the hip trajectory tracking errors.

A first measure in this analysis is the mean absolute density of energy per traveled distance. This index is computed assuming that energy regeneration is not available by actuators doing negative work, that is, by taking the absolute value of the power. At a given joint  $j$  (each leg has  $m = 2$  joints) and leg  $i$  (since we are adopting an hexapod it yields  $n = 6$  legs), the mechanical power is the product of the motor torque and angular velocity. The global index  $E_{av}$  is obtained by averaging the mechanical absolute energy delivered over the traveled distance  $L$ :

$$E_{av} = \frac{1}{L} \sum_{i=1}^n \sum_{j=1}^m \int_0^T \left| \tau_{ijm}(t) \dot{\theta}_{ij}(t) \right| dt. \quad (15)$$

A second measure is the index  $F_L$  that considers the forces that occur on the hips of the robot per traveled distance  $L$ :

$$F_L = \frac{1}{L} \sqrt{\sum_{i=1}^n \int_0^T \{ [f_{ix}(t)]^2 + [f_{iy}(t)]^2 \} dt}. \quad (16)$$

Since the two previous indices capture, mainly, the low frequency system behavior we define the index  $\tau_P$  that measures the robot torque peaks demanded by the controller when responding to high frequency phenomena, such as foot-ground impacts:

$$\tau_P = \max |\tau_{ijC}(t)|. \quad (17)$$

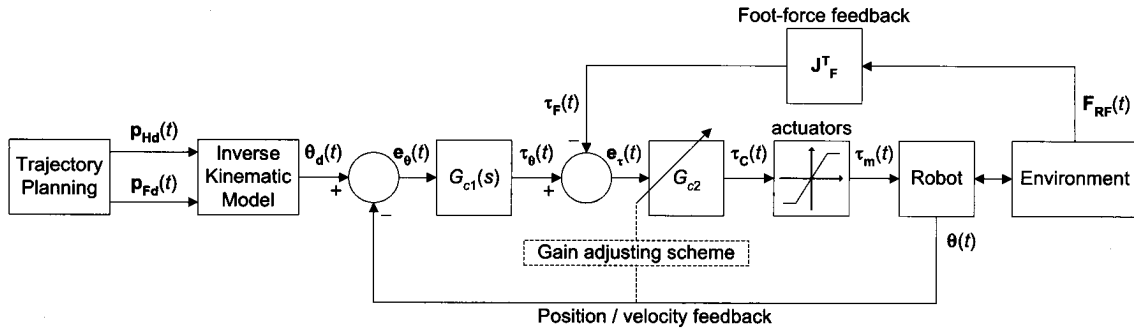


Fig. 3 Hexapod robot control architecture

Table 1 Locomotion parameters

$\beta$	50%
$L_S$	1 m
$H_B$	0.9 m
$F_C$	0.1 m
$V_F$	1 ms <sup>-1</sup>

In what concerns the hip trajectory following errors we can define the index:

$$\varepsilon_{xyH} = \sum_{i=1}^n \sqrt{\frac{1}{N_S} \sum_{k=1}^{N_S} (\Delta_{ixH}^2 + \Delta_{iyH}^2)}, \quad (18)$$

$$\Delta_{ixH} = x_{iH}^d(k) - x_{iH}^r(k),$$

$$\Delta_{iyH} = y_{iH}^d(k) - y_{iH}^r(k)$$

where  $N_S$  is the total number of samples for averaging purposes and  $\{d, r\}$  indicate the  $i$ th samples of the desired and real position, respectively.

In all cases the performance optimization requires the minimization of each index.

## 5. Simulation Results

In this section we develop a set of simulations to compare the controller performances during a periodic Wave gait, while the robot is considered to be traversing a perfectly horizontal surface.

In case of locomotion in terrains with obstacles, the robot should adapt its gait to the environment. If the number of obstacles is small, the locomotion can be accomplished using a quasi periodic gait, for which the trajectory planning using the algorithms described in [26] can be used. In case of more rugged terrains, other planning algorithms are available [27]–[29].

### 5.1 System parameters and controller tuning

For simulation purposes we consider the locomotion parameters, the robot body parameters and the ground parameters (considering that the ground is of compact clay—see Appendix) presented in Tables 1, 2 and 3, respectively.

To tune the controller parameters we adopt a systematic method, testing and evaluating several possible combinations of controller parameters, for both architectures. Moreover, it is assumed high performance joint actuators with a maximum actuator torque in Eq. (9) of  $\tau_{ijMax} = 400$  Nm.

Table 2 Robot model parameters

$S_P$	1 m
$L_{ij}$	0.5 m
$O_i$	0 m
$M_{ij}$	1 kg
$M_b$	88.0 kg
$M_{if}$	0.0 kg
$K_{xH}$	10 <sup>5</sup> Nm <sup>-1</sup>
$K_{yH}$	10 <sup>4</sup> Nm <sup>-1</sup>
$B_{xH}$	10 <sup>3</sup> Nsm <sup>-1</sup>
$B_{yH}$	10 <sup>2</sup> Nsm <sup>-1</sup>

Table 3 Ground parameters

$K_{xF}$	1302152 Nm <sup>-1</sup>
$K_{yF}$	1705199 Nm <sup>-1</sup>
$B'_{xF}$	2364932 Nsm <sup>-1</sup>
$B'_{yF}$	2706233 Nsm <sup>-1</sup>
$v$	0.9

Table 4 Controller parameters for  $G_{c1}(s)$  and  $G_{c2}$  when minimizing the hips trajectories errors  $\varepsilon_{xyH}$

PD-P	$G_{c1}(s)$	Joint $j = 1$	$Kp_{i1}$	1000
			$Kd_{i1}$	100
	$G_{c2}(s)$	Joint $j = 2$	$Kp_{i2}$	4500
			$Kd_{i2}$	20
PD-P&F	$G_{c1}(s)$	Joint $j = 1$	$Kp_{i1}$	1000
			$Kd_{i1}$	100
	$G_{c2}(s)$	Joint $j = 2$	$Kp_{i2}$	4500
			$Kd_{i2}$	20
$G_{c2}(s)$	Joint $j = 1$	$Kp_{i1}$	0.9	
	Joint $j = 2$	$Kp_{i2}$	0.9	

The minimization of the hips trajectories errors ( $\varepsilon_{xyH}$ ) leads to the  $G_{c1}(s)$  and  $G_{c2}$  controller parameters presented in Table 4.

Through the analysis of the plots of Figs. 4 and 5 we conclude that the PD-P&F control architecture improves the

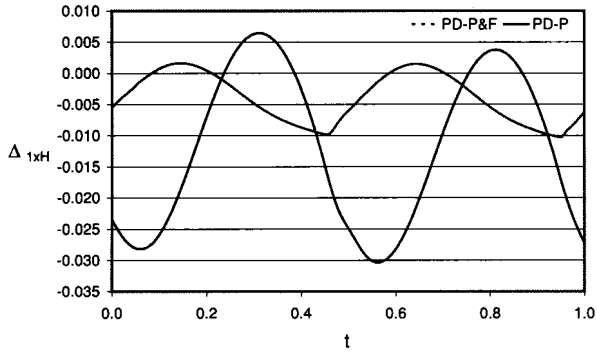


Fig. 4 Plots of the hip trajectory error  $\Delta_{1xH}$  vs.  $t$  for the  $PD-P$  and the  $PD-P\&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

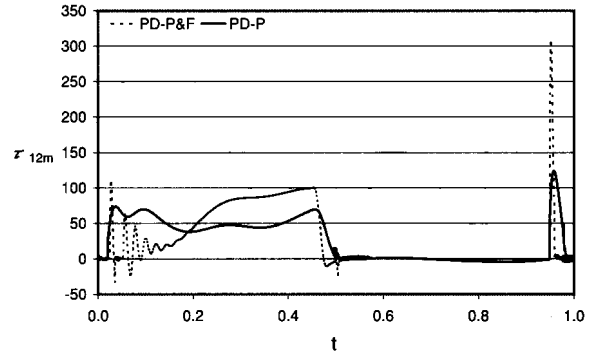


Fig. 7 Plots of the joint torque  $\tau_{12m}$  vs.  $t$  for the  $PD-P$  and the  $PD-P\&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

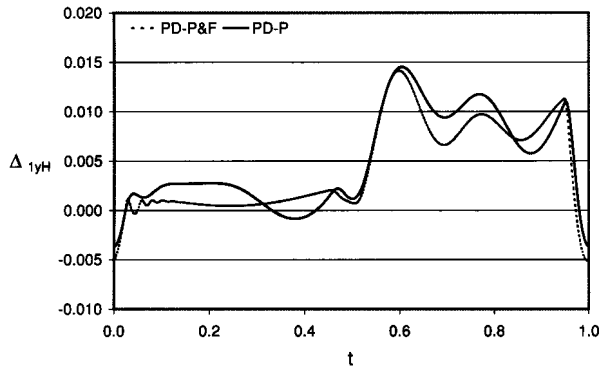


Fig. 5 Plots of the hip trajectory error  $\Delta_{1yH}$  vs.  $t$  for the  $PD-P$  and the  $PD-P\&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

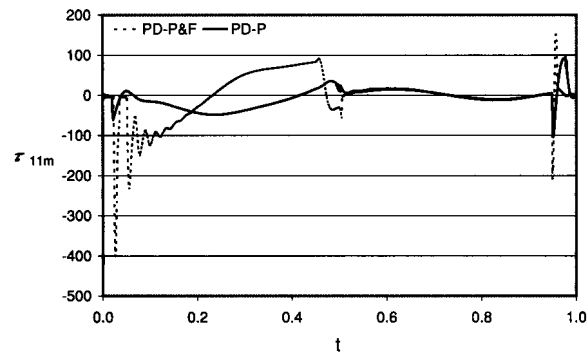


Fig. 6 Plots of the joint torque  $\tau_{11m}$  vs.  $t$  for the  $PD-P$  and the  $PD-P\&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

Table 5 Controller parameters for  $G_{c1}(s)$  and  $G_{c2}$  when establishing a compromise between the minimization of  $E_{av}$  and  $\epsilon_{xyH}$

PD-P	$G_{c1}(s)$	Joint $j = 1$	$Kp_{i1}$	2000
			$Kd_{i1}$	0
	Joint $j = 2$	$Kp_{i2}$	2500	
		$Kd_{i2}$	60	
$G_{c2}(s)$	Joint $j = 1$	$Kp_{i1}$	1.0	
	Joint $j = 2$	$Kp_{i2}$	1.0	
PD-P&F	$G_{c1}(s)$	Joint $j = 1$	$Kp_{i1}$	8000
			$Kd_{i1}$	60
	Joint $j = 2$	$Kp_{i2}$	500	
		$Kd_{i2}$	40	
$G_{c2}(s)$	Joint $j = 1$	$Kp_{i1}$	0.9	
	Joint $j = 2$	$Kp_{i2}$	0.9	

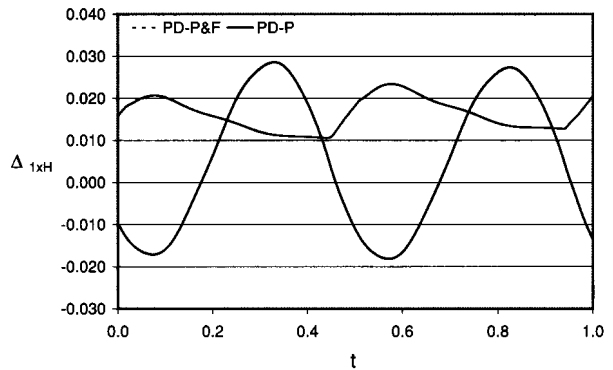


Fig. 8 Plots of the hip trajectory error  $\Delta_{1xH}$  vs.  $t$  for the  $PD-P$  and the  $PD-P\&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

hip trajectory tracking (and presents a lower value for the index  $\epsilon_{xyH}$ ) but at the cost of larger joint driving torques (Figs. 6 and 7) and higher values of  $E_{av}$ ,  $F_L$  and  $\tau_P$ .

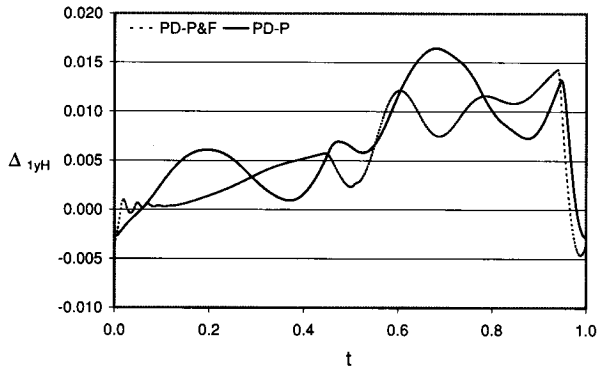
In order to make a proper comparison of both controllers, namely with the variation of the locomotion and the environment parameters, we establish a compromise in what concerns the minimization of  $E_{av}$  and  $\epsilon_{xyH}$ . Therefore, we adopt the  $G_{c1}(s)$  parameters presented in Table 5 and we maintain the value of the proportional controller  $G_{c2}$  gain  $Kp_j$ .

For this new set of controller parameters we conclude again that the  $PD-P\&F$  improves the hip trajectory tracking (Figs. 8 and 9), while presenting similar joint torques (Figs. 10 and 11) to the  $PD-P$ . Furthermore, in the case of the  $PD-P$  the joint torque  $\tau_{12m}$  presents a considerable os-

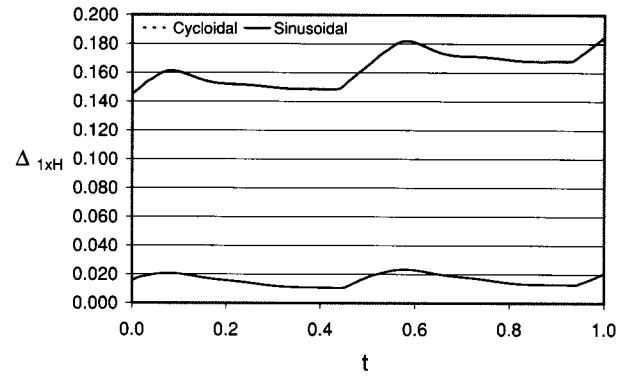
cillation at the instants of feet impact in the ground (which occur around  $t = 0.5$  s) and a more subtle oscillation during the foot support phase ( $0.5$  s  $<$   $t$   $<$   $1.0$  s).

### 5.2 Foot trajectory and actuator saturation

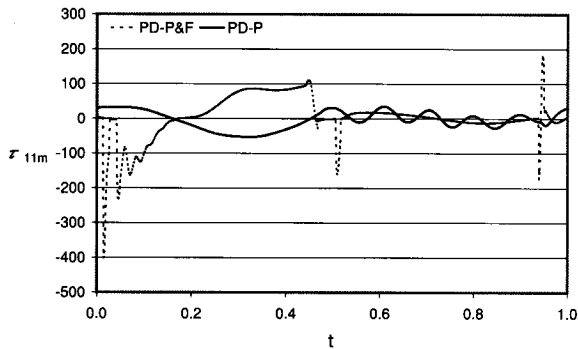
With the robot, ground and controller parameters of Tables 1–3 and 5 we evaluate the two alternative foot transfer trajectories  $p_{Fd}(t)$ , namely the cycloidal and the sinusoidal functions, presented in Eqs. (1) and (2). From Figs. 12 and 13 it is possible to conclude that the cycloidal foot transfer function is superior because it improves the hip tra-



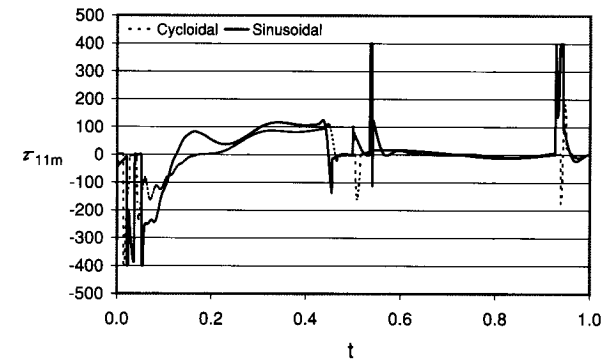
**Fig. 9** Plots of the hip trajectory error  $\Delta_{1yH}$  vs.  $t$  for the *PD-P* and the *PD-P&F* control architectures, with  $\tau_{ijMax} = 400$  Nm



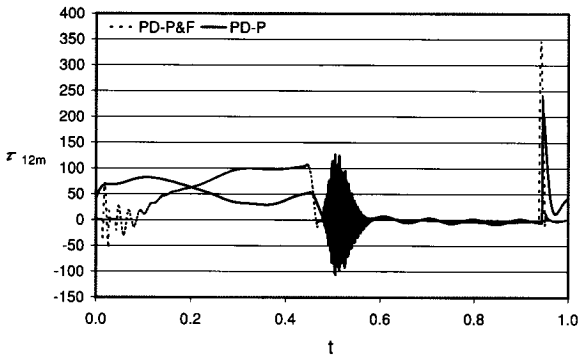
**Fig. 12** Plots of the hip trajectory error  $\Delta_{1xH}(t)$  for the *cycloidal* and *sinusoidal* foot trajectories, with the *PD-P&F* controller and  $\tau_{ijMax} = 400$  Nm



**Fig. 10** Plots of the joint torque  $\tau_{11m}$  vs.  $t$  for the *PD-P* and the *PD-P&F* control architectures, with  $\tau_{ijMax} = 400$  Nm



**Fig. 13** Plots of the joint torque  $\tau_{11m}(t)$  for the *cycloidal* and *sinusoidal* foot trajectories, with the *PD-P&F* controller and  $\tau_{ijMax} = 400$  Nm



**Fig. 11** Plots of the joint torque  $\tau_{12m}$  vs.  $t$  for the *PD-P* and the *PD-P&F* control architectures, with  $\tau_{ijMax} = 400$  Nm

jectory tracking, while minimizing the corresponding joint torques. Furthermore, in the case of the sinusoidal foot transfer trajectory, the joint torques present large amplitude spikes in the instants of feet take-off and landing, which can be observed for  $\tau_{11m}$  around  $t = 0, 0.5$  and  $1.0$  s in Fig. 13. These spikes are due to some dragging of the feet on the ground caused by the shape of the sinusoidal function. For different acceleration profiles of the foot trajectory there were no significant changes of these charts. Based on this conclusion, in the sequel we adopt only the cycloidal foot trajectory when testing the control algorithms.

Another important aspect is the required actuator action. Therefore, we decide to test both controller performances when varying the level of actuator saturation ( $\tau_{ijMax}$ ). The *PD-P&F* controller architecture presents lower values of

$\varepsilon_{xyH}$  for values of  $\tau_{ijMax} > 100$  Nm. On the other hand, both controllers present similar values of  $E_{av}$  in this interval. In the case of strong actuator saturation level ( $\tau_{ijMax} < 100$  Nm) the two algorithms have difficulties in controlling the robot locomotion and, as a consequence, the values of the indices  $\varepsilon_{xyH}$  and  $E_{av}$  increase sharply, being this effect slightly higher in the case of the *PD-P&F* controller.

During the feet collisions with the ground, at the instants of feet landing, high contact forces occur that propagate through the robot legs, leading to large values of the index  $F_L$ . Moreover, those contact forces feed-back in the controller force loop leading to large values of the index  $\tau_P$ , for the *PD-P&F* scheme. For this reason, the *PD-P* controller architecture presents lower values for the indices  $F_L$  and  $\tau_P$  for the entire range of  $\tau_{ijMax}$ .

### 5.3 Locomotion parameters

An important aspect that must be investigated is the influence of the locomotion parameters. Therefore, we test the two control architectures for variations of the duty factor  $\beta$ , the robot forward locomotion velocity  $V_F$  and the locomotion gait.

The duty factor was varied within the interval capable of achieving a stable locomotion ( $20\% \leq \beta \leq 80\%$ ). The four indices are lower in the case of the *PD-P&F* architecture, for all values of  $\tau_{ijMax}$ .

Regarding the variation of the robot velocity, the results were analyzed for  $0.5 \text{ ms}^{-1} < V_F < 6.0 \text{ ms}^{-1}$ .

For the entire range of variation of  $V_F$ , the  $PD-P&F$  control scheme presents the lower values for  $\varepsilon_{xyH}$ . Concerning the dynamical indices, for  $\tau_{ijMax} > 100$  Nm, their behaviour is similar and we have superior performances for the  $PD-P&F$  scheme if  $V_F < 1.0$  ms<sup>-1</sup> or  $V_F > 3.0$  ms<sup>-1</sup> and for the  $PD-P$  architecture if  $1.0$  ms<sup>-1</sup> <  $V_F$  <  $3.0$  ms<sup>-1</sup>.

In what concerns the locomotion parameters, we finally tested the controllers performance under different locomotion gaits, namely with Equal Phase Half Cycle, Equal Phase Full Cycle, Backward Wave, Backward Equal Phase Half Cycle and Backward Equal Phase Full Cycle [2], for values of the duty factor capable of achieving a stable locomotion ( $20\% \leq \beta \leq 80\%$ ). The results show that, for both control schemes, the best performance is attained for the Wave gait, being the results quite similar to the Backward Wave gait. Concerning the other four gaits, the performance indices are similar for all of them.

### 5.4 Ground properties

Based on the experiments presented previously we examine both control architectures for distinct ground properties. Therefore, in a first phase we start by considering the  $PD-P$  controller and different values of  $(K_{xF}, K_{yF}, B_{xF}, B_{yF})$ , in order to observe its influence upon the proposed indices, for actuators such that  $\tau_{ijMax} = 400$  Nm. In a second phase we repeat the experiments for the case of a  $PD-P&F$  control architecture.

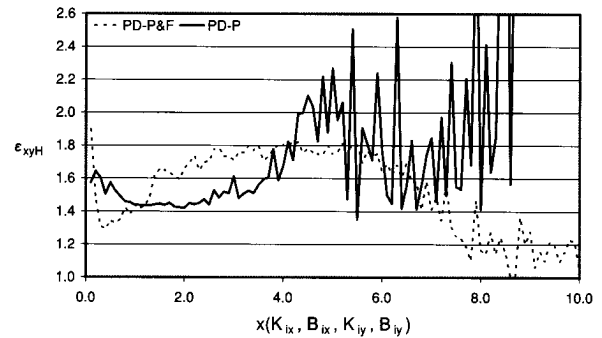
The range of ground model parameters allow us to simulate the behavior of the ground from peat to gravel. For ground parameters similar to those of concrete and wood, both algorithms present difficulties in controlling the robot locomotion with the adopted parameters, due to the impacts that appear during the feet landing. In this case there are two options: we can find another tuning for the controllers or we can equip the robot feet with some shock absorbing device [30]–[33].

The performance measures *versus* the multiplying coefficient of ground parameters, with relation to base experiment, are presented in **Figs. 14–17**. We conclude that the robot hips trajectories errors (measured through the index  $\varepsilon_{xyH}$ ) are smaller when we adopt a  $PD-P&F$  control architecture, for values of the ground parameters below those adopted in the base experience (soft grounds) and above  $4 \times$  the base parameter values (hard soils), for which case the  $PD-P$  control architecture finds problems in correctly controlling the system.

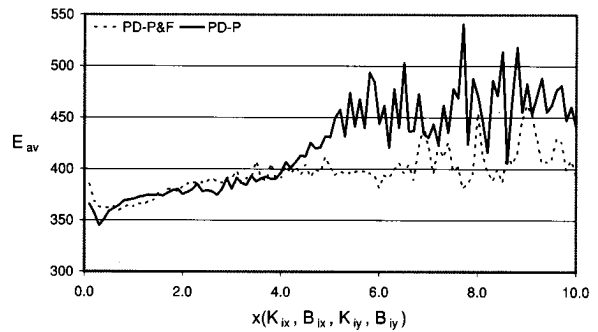
The behavior of the index  $E_{av}$  is similar to  $\varepsilon_{xyH}$ . Concerning the indices  $F_L$  and  $\tau_P$  they are higher in the case of the  $PD-P&F$  control architecture for all types of ground.

For moderate levels of actuator saturation (e.g.,  $\tau_{ijMax} = 150$  Nm), we get similar conclusions.

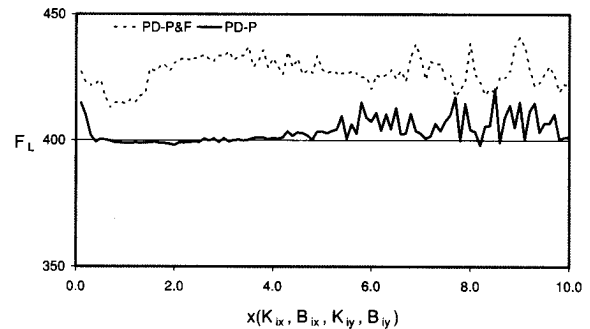
In the case of strong actuator saturation (e.g.,  $\tau_{ijMax} < 140$  Nm) the indices reveal a large performance degradation with difficulties both for the  $PD-P&F$  and the  $PD-P$  controllers. Nevertheless, this situation is not realistic since it corresponds to operating conditions requiring joint torques much higher than those established by the saturation level. On the other hand, when we have high performance actuators ( $\tau_{ijMax} > 400$  Nm), the  $PD-P&F$  scheme reveals stability problems, particularly on hard ter-



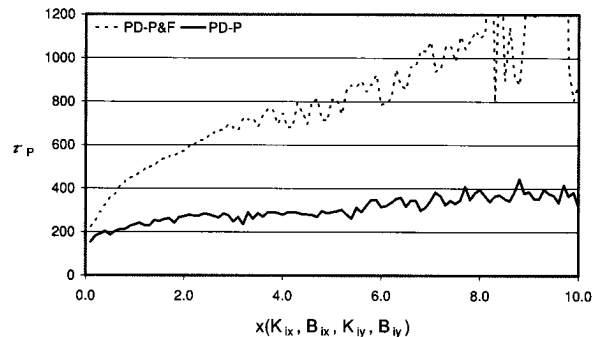
**Fig. 14** Plot of  $\varepsilon_{xyH}$  vs.  $\times(K_{xF}, K_{yF}, B_{xF}, B_{yF})$  for the  $PD-P$  and the  $PD-P&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm



**Fig. 15** Plot of  $E_{av}$  vs.  $\times(K_{xF}, K_{yF}, B_{xF}, B_{yF})$  for the  $PD-P$  and the  $PD-P&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

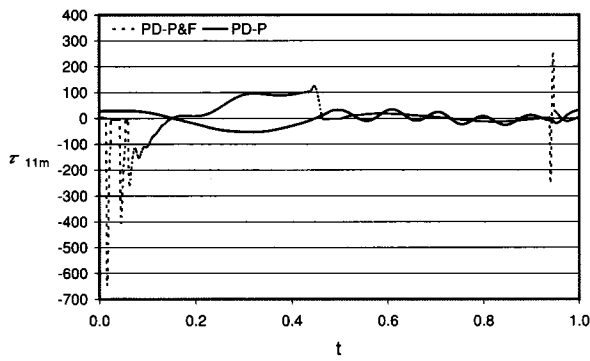


**Fig. 16** Plot of  $F_L$  vs.  $\times(K_{xF}, K_{yF}, B_{xF}, B_{yF})$  for the  $PD-P$  and the  $PD-P&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

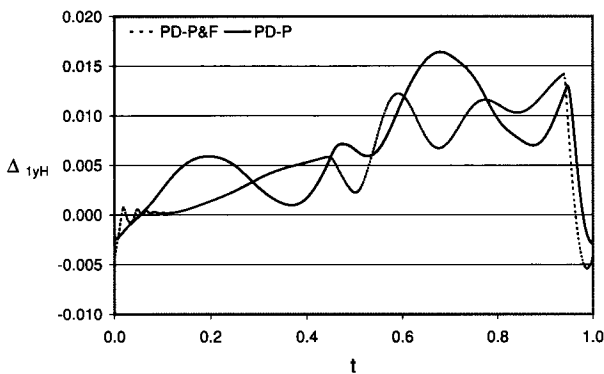


**Fig. 17** Plot of  $\tau_P$  vs.  $\times(K_{xF}, K_{yF}, B_{xF}, B_{yF})$  for the  $PD-P$  and the  $PD-P&F$  control architectures, with  $\tau_{ijMax} = 400$  Nm

rains (values of the ground parameters above 100% of the base values) due to the impulses of force feedback during the impacts of the feet with the ground (**Figs. 18 and 19**).



**Fig. 18** Plots of the of the joint torque  $\tau_{11m}$  vs.  $t$  for the  $PD-P$  and the  $PD-P\&F$  control architectures, with  $\tau_{ijMax} \rightarrow \infty$



**Fig. 19** Plots of the hip trajectory error  $\Delta_{1yH}$  vs.  $t$  for the  $PD-P$  and the  $PD-P\&F$  control architectures, with  $\tau_{ijMax} \rightarrow \infty$

However, this case is also not realistic since it assumes ideal actuators exhibiting infinite joint driving torque and infinite bandwidth.

In conclusion, the foot-force feedback seems essential for a robust control performance during walking in terrain with variable dynamical characteristics.

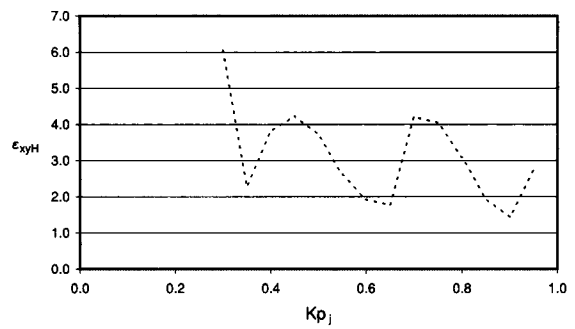
### 5.5 Adaptive scheme for the $G_{c2}$ controller gains

An aspect that must be evaluated for the  $PD-P\&F$  scheme is the effect of  $G_{c2}$  upon its performance. **Figures 20–22** reveal that a larger proportional gain  $Kp_j$  ( $j = 1, 2$ ) leads to smaller  $\varepsilon_{xyH}$ ,  $E_{av}$  and  $F_L$  but, on the other hand, degrades the index  $\tau_P$  (**Fig. 23**). Moreover, high  $G_{c2}$  gains lead to large feet-ground impacts that propagate through the control loop up to the actuators, as previously mentioned.

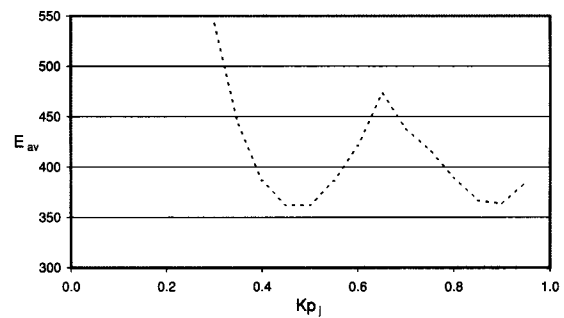
The results point out that the main problem with the  $PD-P\&F$  scheme are the high gains in the forward loop, leading to large feet-ground impacts that propagate through the control loop up to the actuators, at the instants of feet landing.

Bearing these facts in mind, we establish a gain scheduling policy that varies the gain adaptively in accordance with the feet relative position to the ground. The idea is to reduce the proportional gains  $Kp_j$  ( $j = 1, 2$ ), of the  $G_{c2}$  controller, before the instants of feet landing and to increase them again after the feet take-off.

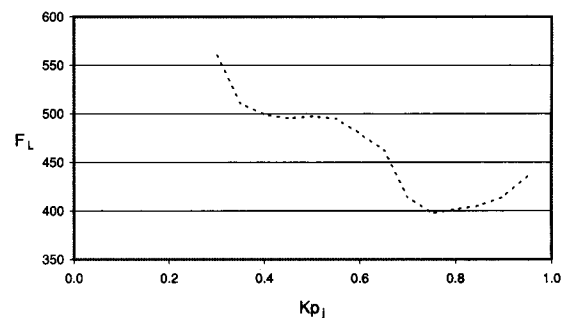
**Figures 24–27** depict the four indices *versus* the value of the gains  $Kp_j$  of the  $G_{c2}$  controller when the feet are in the



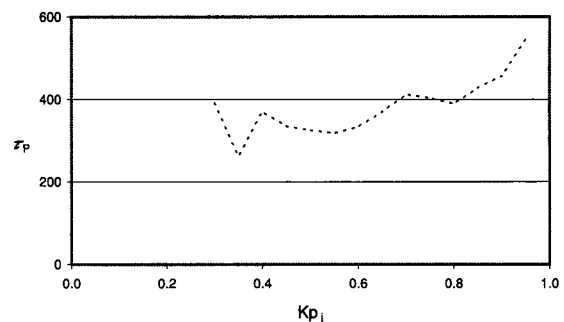
**Fig. 20** Plot of  $\varepsilon_{xyH}$  vs.  $Kp_j$  for the  $PD-P\&F$  controller, with  $\tau_{ijMax} = 400$  Nm



**Fig. 21** Plot of  $E_{av}$  vs.  $Kp_j$  for the  $PD-P\&F$  controller, with  $\tau_{ijMax} = 400$  Nm

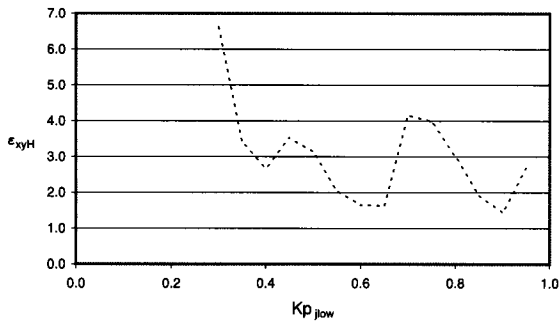


**Fig. 22** Plots of  $F_L$  vs.  $Kp_j$  for the  $PD-P\&F$  controller, with  $\tau_{ijMax} = 400$  Nm

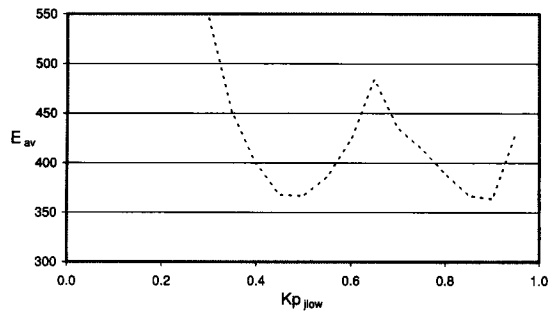


**Fig. 23** Plot of  $\tau_P$  vs.  $Kp_j$  for the  $PD-P\&F$  controller, with  $\tau_{ijMax} = 400$  Nm

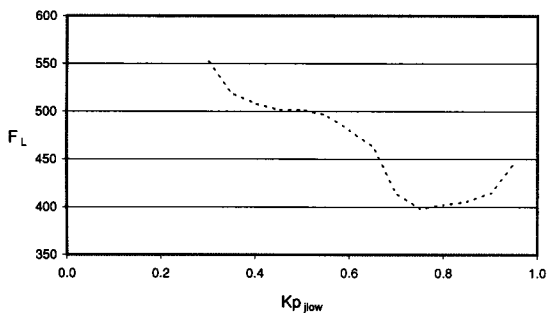
support phase ( $Kp_{jlow}$ ). During the feet transfer phase the value of these gains is  $Kp_{jhigh} = 0.9$ . The switching between the support and transfer phases is established when



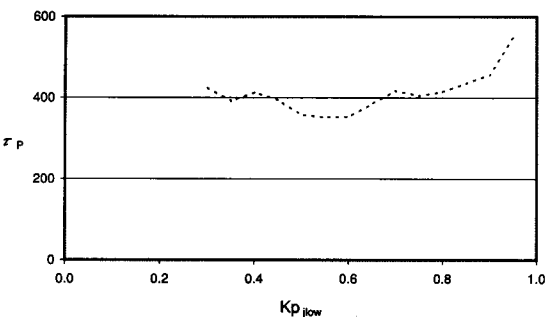
**Fig. 24** Plot of  $\epsilon_{xyH}$  vs.  $Kp_{low}$  for  $\{Kp_{jhigh}, y_{iF0}\} = \{0.9, 0.01\}$ , with the PD-P&F controller and  $\tau_{ijMax} = 400$  Nm



**Fig. 25** Plot of  $E_{av}$  vs.  $Kp_{low}$  for  $\{Kp_{jhigh}, y_{iF0}\} = \{0.9, 0.01\}$ , with the PD-P&F controller and  $\tau_{ijMax} = 400$  Nm



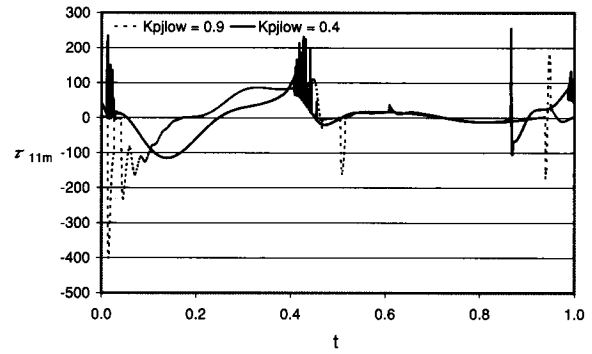
**Fig. 26** Plots of  $F_L$  vs.  $Kp_{low}$  for  $\{Kp_{jhigh}, y_{iF0}\} = \{0.9, 0.01\}$ , with the PD-P&F controller and  $\tau_{ijMax} = 400$  Nm



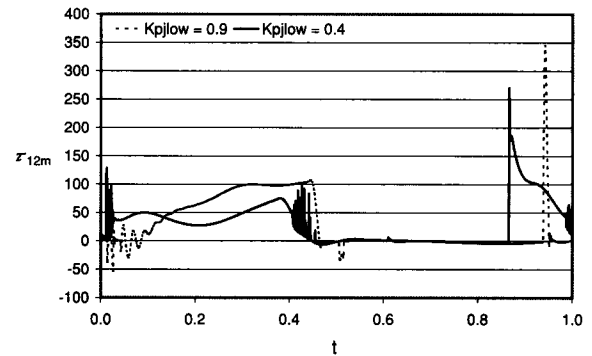
**Fig. 27** Plot of  $\tau_P$  vs.  $Kp_{low}$  for  $\{Kp_{jhigh}, y_{iF0}\} = \{0.9, 0.01\}$ , with the PD-P&F controller and  $\tau_{ijMax} = 400$  Nm

the distance of the feet to the ground is  $y_{iF0} = 0.01$ .

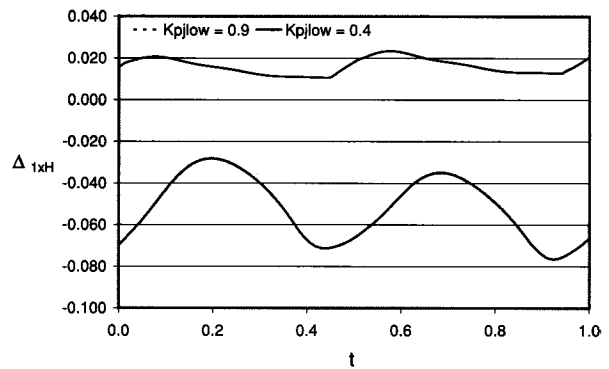
Figure 24 reveals that increasing the proportional gain



**Fig. 28** Plots of the joint torque  $\tau_{11m}$  vs.  $t$  for  $\{Kp_{jhigh}, y_{iF0}\} = \{0.9, 0.01\}$ , with the PD-P&F controller and  $\tau_{ijMax} = 400$  Nm



**Fig. 29** Plots of the joint torque  $\tau_{12m}$  vs.  $t$  for  $\{Kp_{jhigh}, y_{iF0}\} = \{0.9, 0.01\}$ , with the PD-P&F controller and  $\tau_{ijMax} = 400$  Nm

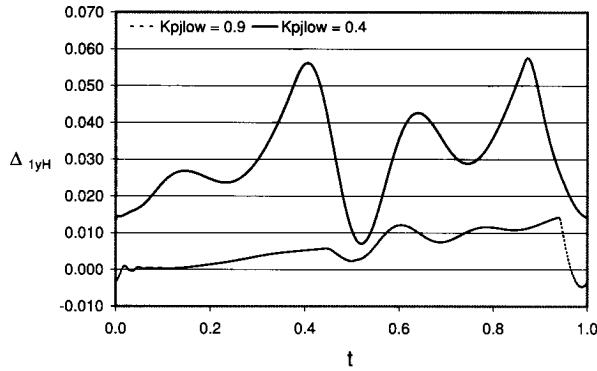


**Fig. 30** Plots of the hip trajectory error  $\Delta_{1xH}$  vs.  $t$  for  $\{Kp_{jhigh}, y_{iF0}\} = \{0.9, 0.01\}$ , with the PD-P&F controller and  $\tau_{ijMax} = 400$  Nm

up to  $Kp_{low} = 0.9$  leads to a better following of the hips trajectories and improves the indices  $E_{av}$  and  $F_L$  (Figs. 25 and 26, respectively) but, on the other hand, degrade the index  $\tau_P$  (Fig. 27).

Based on the analysis of these charts, we can find a compromise situation. An adaptive scheme such that  $Kp_j(y_{iF}) = Kp_{low}$  if  $y_{iF} \leq y_{iF0}$  or  $Kp_j(y_{iF}) = Kp_{jhigh}$  if  $y_{iF} > y_{iF0}$  for  $\{Kp_{jlow}, Kp_{jhigh}, y_{iF0}\} = \{0.4, 0.9, 0.01\}$  leads to a good compromise between the trajectory following errors and the dynamic indices, as can be seen in Figs. 28–31.

In conclusion, the foot-force feedback with a reduction



**Fig. 31** Plots of the hip trajectory error  $\Delta_{1yH}$  vs.  $t$  for  $\{K_{p_{high}}, y_{iF0}\} = \{0.9, 0.01\}$ , with the *PD-P&F* controller and  $\tau_{ijMax} = 400$  Nm

of the proportional gains of the forward loop during the support phase seems essential for establishing a robust control during walking and to accommodate foot-ground interaction phenomena.

## 6. Conclusions

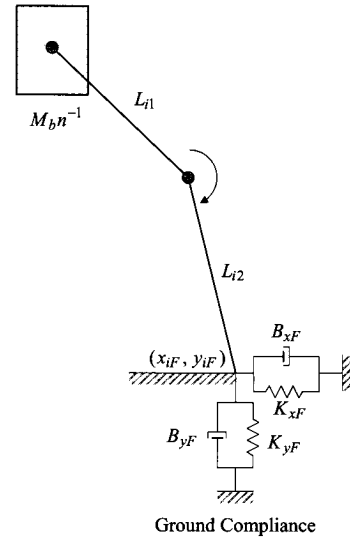
In this paper we have compared the performance of *PD* control algorithms with position or position and force feedback, in hexapod robots, for different locomotion parameters. Furthermore, we evaluated how the different robot controller architectures respond to non-ideal joint actuators, namely with torque saturation, and variable ground dynamic properties.

For analyzing the system performance four quantitative measures were defined based on the system dynamics and the hip trajectory errors. The experiments reveal that the *PD-P&F* control architecture (cascade controller) is superior to the classical *PD-P* control scheme, from the point of view of the proposed indices. They also show that the cascade controller, with the foot-force feedback and a reduction of the proportional gains of the forward force loop during the support phase, is superior, from the point of view of the proposed indices.

## 7. Appendix

The contact of the robot feet with the ground can be analyzed through different approaches leading to distinct models. On one hand, it is possible to use the exact force-deflection relationships. On the other hand, and under specific restrictions, it is possible to use approximate models of the ground deformation based on the studies of soil mechanics.

One example of the first approach was used by Manko [1], that models the foot-ground interactions, using force-deflection relationships, for different loading conditions on flat and sloped surfaces. Manko uses a bilinear equation for modeling vertical foot-ground interactions while the lateral forces are modeled with an expression describing an exponential transition to Coulomb's equation. Another example was given by Bekker [34] that relates the vertical sinkage and the local pressure normal to the ground surface through an exponential function. Bekker relates the horizontal deflection with the local shear stress at the ground surface using a quotient of exponential functions.



**Fig. 32** Mass-Spring-Dashpot model of the foot-ground interaction

The second approach models the foot-ground interaction through a linear system (**Fig. 32**) with damping  $B_{\eta F}$  and stiffness  $K_{\eta F}$  ( $\eta = \{x, y\}$ ) in the {horizontal, vertical} directions, respectively, according to equations [35]:

$$\begin{aligned} f_{ixF} &= -K_{xF} (\Delta_{ixF}) - B_{xF} (\dot{\Delta}_{ixF}), \\ \Delta_{ixF} &= x_{iF} - x_{iF0}, \quad \dot{\Delta}_{ixF} = \dot{x}_{iF} - \dot{x}_{iF0} \end{aligned} \quad (19)$$

$$\begin{aligned} f_{iyF} &= -K_{yF} (\Delta_{iyF}) - B_{yF} (\dot{\Delta}_{iyF}), \\ \Delta_{iyF} &= y_{iF} - y_{iF0}, \quad \dot{\Delta}_{iyF} = \dot{y}_{iF} - \dot{y}_{iF0} \end{aligned} \quad (20)$$

where  $x_{iF0}$  and  $y_{iF0}$  are the coordinates of foot  $i$  touchdown.

The parameters of these equations are given by [35]:

$$K_{xF} = 2(1 + \mu) G \beta_x \sqrt{BL} \quad (21)$$

$$K_{yF} = \frac{G}{1 - \mu} \beta_y \sqrt{BL} \quad (22)$$

where  $\mu$  is the Poisson's ratio (whose value varies among 0.35 for soils of low saturation and 0.5 for fully saturated soils [35]),  $B$  is the width of the feet touching the ground and  $L$  the corresponding length. The values of  $\beta_x$  and  $\beta_y$  are extracted from pre-calculated tables given in [35] as functions of  $L/B$ . Finally  $G$  is the shear modulus and is calculated using the following expression:

$$G = \frac{E}{2(1 + \mu)} \quad (23)$$

where  $E$  is the Young's modulus of elasticity of the soil type. The Young's modulus of elasticity of some common soil types are indicated in **Table 6** (extracted from [36]).

Concerning the values of damping ( $B_{xF}$  and  $B_{yF}$ ) these can be calculated considering that the above expressions apply to a Mass-Spring-Dashpot system whose damping ratios are extracted from pre-calculated tables given in [35] as functions of  $b$ , being:

$$b = \frac{M}{\rho R^3} \quad (24)$$

**Table 6** Young's moduli of different soil types

Soil Type	Young's Modulus (kNm <sup>-2</sup> )
Concrete	30000000
Wood	13000000
Gravel	100000 – 200000
Sand	10000 – 80000
Compact Clay	3000 – 15000
Loose Clay	500 – 3000
Peat	100 – 500

**Table 7** Ground model parameters for different soil types

Soil Type	$K_{xF}$	$B_{xF}$	$K_{yF}$	$B_{yF}$
Concrete	2604304130	153097	3410398265	175196
Wood	1128531790	100781	1477839248	115328
Gravel	17362028	12500	22735988	14305
Sand	6944811	7906	9094395	9047
Compact Clay	1302152	3423	1705199	3917
Loose Clay	260430	1531	341040	1752
Peat	43405	625	56840	715

where  $M$  is the equivalent mass of the system under consideration (in our case  $M_b n^{-1}$ ),  $\rho$  is the mass density of the soil [35] [36] and  $R$  is the radius of the feet-ground contact area. For the typical dimensions of the robot under consideration, the ground damping ratio either in the horizontal and vertical directions is  $\zeta < 0.15$ .

Based on the above theory, **Table 7** presents values for the ground model parameters of typical soils commonly found in nature and in the living environments.

In order to convert the parameters of this linear foot-ground interaction model ( $B_{xF}$ ,  $B_{yF}$ ) to the parameters of the non-linear model ( $B'_{xF}$ ,  $B'_{yF}$ ), described by Eqs. (10) and (11), we use the following relations:

$$-B'_{xF} (-\Delta_{iyFM\alpha x}) = -B_{xF} \quad (25)$$

$$-B'_{yF} (-\Delta_{iyFM\alpha x})^v = -B_{yF} \quad (26)$$

where  $\Delta_{iyFM\alpha x}$  is the maximum depth that the robot feet penetrates the ground.

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## Biographies

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